

Dymore User's Manual

Definition of mass properties

1 The parallel axis theorem

Figure 1 depicts the configuration of the rigid body: \underline{s} is the position vector of a material point \mathbf{Q} of the rigid body with respect to reference point \mathbf{B} , and \underline{q} is the position vector of the same point with respect to the center of mass \mathbf{C} . Clearly, $\underline{s} = \underline{\eta} + \underline{q}$, where $\underline{\eta}$ is the position vector of the center of mass with respect to point \mathbf{B} .

The tensor of mass moments of inertia evaluated with respect to point \mathbf{B} is now

$$\underline{\underline{\varrho}}^B = \int_{\mathcal{V}} \tilde{s} \tilde{s}^T \rho d\mathcal{V} = \int_{\mathcal{V}} (\tilde{\eta} + \tilde{q})(\tilde{\eta}^T + \tilde{q}^T) \rho d\mathcal{V},$$

where \mathcal{V} is the volume of the rigid body. Expanding the integrand and taking advantage of the fact that position vector $\underline{\eta}$ can be factored out of the integral sign leads to

$$\underline{\underline{\varrho}}^B = m \tilde{\eta} \tilde{\eta}^T + \tilde{\eta} \left[\int_{\mathcal{V}} \tilde{q}^T \rho d\mathcal{V} \right] + \left[\int_{\mathcal{V}} \tilde{q} \rho d\mathcal{V} \right] \tilde{\eta}^T + \int_{\mathcal{V}} \tilde{q} \tilde{q}^T \rho d\mathcal{V}.$$

The two middle terms vanish because $\int_{\mathcal{V}} \tilde{q} \rho d\mathcal{V} = m \tilde{r}_{CC} = 0$. The last term is the mass moment of inertia tensor, $\underline{\underline{\varrho}}^C$, evaluated with respect to the center of mass, and hence,

$$\underline{\underline{\varrho}}^B = \underline{\underline{\varrho}}^C + m \tilde{\eta} \tilde{\eta}^T. \quad (1)$$

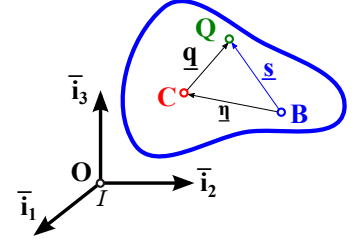


Figure 1: Evaluating the mass moments of inertia with respect to a reference point \mathbf{B} and the center of mass \mathbf{C} .

Let ϱ_{ij}^B and ϱ_{ij}^C be the components of the mass moment of inertia tensors $\underline{\underline{\varrho}}^B$ and $\underline{\underline{\varrho}}^C$, respectively, and let $\{\eta_1, \eta_2, \eta_3\}^T$ be the components of vector $\underline{\eta}$, all resolved in the same basis. The diagonal components of tensor $\underline{\underline{\varrho}}^B$ now become

$$\varrho_{11}^B = \varrho_{11}^C + m(\eta_2^2 + \eta_3^2), \quad (2a)$$

$$\varrho_{22}^B = \varrho_{22}^C + m(\eta_1^2 + \eta_3^2), \quad (2b)$$

$$\varrho_{33}^B = \varrho_{33}^C + m(\eta_1^2 + \eta_2^2). \quad (2c)$$

Components ϱ_{11}^B and ϱ_{11}^C of the mass moment of inertia tensor are evaluated with respect to two different points, an arbitrary point \mathbf{B} and the center of mass, respectively, but in the same basis, *i.e.*, with respect to parallel axis systems; hence, the name of *parallel axes theorem*.

The properties of the center of mass were used in the derivation of this theorem, hence, it is incorrect to write $\varrho_{11}^B = \varrho_{11}^R + m(\eta_2^2 + \eta_3^2)$ if points \mathbf{B} and \mathbf{R} are two arbitrary points of the rigid body.

Because the second term on the right-hand side of eqs. (2) is strictly positive, it follows $\varrho_{11}^B > \varrho_{11}^C$, that is, the moment of inertia always increases when moving away from the center of mass. In other words, the minimum value of ϱ_{11} is obtained when it is computed with respect to the center of mass.

The off-diagonal terms of tensor of moments of inertia are called *products of inertia*; in view of eq. (1), they become

$$\varrho_{23}^B = \varrho_{23}^C - m\eta_2\eta_3, \quad (3a)$$

$$\varrho_{13}^B = \varrho_{13}^C - m\eta_1\eta_3, \quad (3b)$$

$$\varrho_{12}^B = \varrho_{12}^C - m\eta_1\eta_2. \quad (3c)$$

In this case, the second term on the right-hand side could be positive or negative; consequently, products of inertia could increase or decrease when moving away from the center of mass.

2 Summary

Consider the components of the moments of inertia tensor resolved in **the local coordinate system**. Let the components of this tensor with respect to a **material point of the body** and with respect to the **center of mass of the body** be denoted

$$\underline{\underline{\rho}}^B = \begin{bmatrix} \rho_{11}^B & \rho_{12}^B & \rho_{13}^B \\ & \rho_{22}^B & \rho_{23}^B \\ & & \rho_{33}^B \end{bmatrix} \quad \text{and} \quad \underline{\underline{\rho}}^C = \begin{bmatrix} \rho_{11}^C & \rho_{12}^C & \rho_{13}^C \\ & \rho_{22}^C & \rho_{23}^C \\ & & \rho_{33}^C \end{bmatrix}, \quad (4)$$

respectively. The relationship between the components of these two tensors is readily found as

$$\underline{\underline{\rho}}^B = \underline{\underline{\rho}}^C + m \begin{bmatrix} \eta_2^2 + \eta_3^2 & -\eta_1\eta_2 & -\eta_1\eta_3 \\ -\eta_1\eta_2 & \eta_1^2 + \eta_3^2 & -\eta_2\eta_3 \\ -\eta_1\eta_3 & -\eta_2\eta_3 & \eta_1^2 + \eta_2^2 \end{bmatrix}, \quad (5)$$

where

$$\underline{\underline{\eta}} = \begin{Bmatrix} \eta_1 \\ \eta_2 \\ \eta_3 \end{Bmatrix}. \quad (6)$$